https://mchba.github.io/240731 intro to earsm.pdf



Slides are here!



# Introduction to EARS models

Mads Baungaard University of Oxford July 31, 2024

#### EARS



#### Explicit Algebraic Reynolds Stress models



# What is an EARS model?

• A class of turbulence models for RANS



Gatski & Jongen (2000)

## Example of an EARS model

• Model of Wallin & Johansson (2000) with  $k-\varepsilon$  platform.

 $\frac{\partial k}{\partial t} + U_j \frac{\partial k}{\partial x_j} = \underbrace{-\overline{u_i' u_j'}}_{\mathcal{P}} \frac{\partial U_i}{\partial x_j} - \varepsilon + \underbrace{\frac{\partial}{\partial x_j} \left(\frac{\nu_t}{\sigma_k} \frac{\partial k}{\partial x_j}\right)}_{\mathcal{D}^{(k)}},$  $\frac{\partial \varepsilon}{\partial t} + U_j \frac{\partial \varepsilon}{\partial x_j} = (C_{\varepsilon 1} \mathcal{P} - C_{\varepsilon 2} \varepsilon) \frac{\varepsilon}{k} + \underbrace{\frac{\partial}{\partial x_j} \left(\frac{\nu_t}{\sigma_\varepsilon} \frac{\partial \varepsilon}{\partial x_j}\right)}_{\mathcal{D}^{(\varepsilon)}}.$ 

$$\mathbf{S} = S_{ij} = \frac{1}{2} \frac{k}{\varepsilon} \left( \frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i} \right),$$
$$\mathbf{\Omega} = \Omega_{ij} = \frac{1}{2} \frac{k}{\varepsilon} \left( \frac{\partial U_i}{\partial x_j} - \frac{\partial U_j}{\partial x_i} \right),$$

$$II_{S} = S_{ij}S_{ji} \qquad \mathbf{T}^{(1)} = \mathbf{S}$$
$$II_{\Omega} = \Omega_{ij}\Omega_{ji} \qquad \mathbf{T}^{(4)} = \mathbf{S}\mathbf{\Omega} - \mathbf{\Omega}\mathbf{S}$$

$$P_{1} = \left(\frac{1}{27}c_{1}^{\prime 2} + \frac{9}{20}II_{S} - \frac{2}{3}II_{\Omega}\right)c_{1}^{\prime},$$

$$P_{2} = P_{1}^{2} - \left(\frac{1}{9}c_{1}^{\prime 2} + \frac{9}{10}II_{S} + \frac{2}{3}II_{\Omega}\right)^{3}.$$

$$N = \begin{cases} \frac{c_{1}^{\prime}}{3} + \left(P_{1} + \sqrt{P_{2}}\right)^{1/3} + \operatorname{sign}\left(P_{1} - \sqrt{P_{2}}\right)|P_{1} - \sqrt{P_{2}}|^{1/3}, & P_{2} \ge 0\\ \frac{c_{1}^{\prime}}{3} + 2\left(P_{1}^{2} - P_{2}\right)^{1/6} + \cos\left(\frac{1}{3}\left(\frac{P_{1}}{\sqrt{P_{1}^{2} - P_{2}}}\right)\right), & P_{2} < 0 \end{cases}$$

$$\beta_{1} = -\frac{6}{5} \frac{N}{N^{2} - 2II_{\Omega}}, \qquad \beta_{4} = -\frac{6}{5} \frac{1}{N^{2} - 2II_{\Omega}}.$$
$$a_{ij} = \beta_{1}T_{ij}^{(1)} + \beta_{4}T_{ij}^{(4)}$$
$$\overline{u'_{i}u'_{j}} = ka_{ij} + \frac{2}{3}k\delta_{ij}$$

# Why EARS models?

	Linear EVM	WJ-EARSM
Anisotropic freestream turbulence	×	
Secondary flows	×	
Counter-gradient heat fluxes	×	
Realizable turbulence	Some	
Sensitive to rotation	Very few	$\checkmark$

Wallin & Johansson (2000):

An explicit algebraic Reynolds stress model

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cases, here with almost one million grid points. Most importantly, there is no substantial increase in the computational cost compared to the, in many cases, robust standard  $K-\omega$  model.

#### How I got interested in EARS models

FLOW centre KTH, Stockholm, Sweden



Visited dr. Stefan Wallin in Autumn 2021.

# EARSM for wind applications

Research article | 🞯 🛈

#### Wind turbine wake simulation with explicit algebraic Reynolds stress modeling

Mads Baungaard  $\square$ , Stefan Wallin, Maarten Paul van der Laan, and Mark Kelly

1	2D WJ-	EARS	4	
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10 Oct 2022

Turbulence modeling for wind turbine wakes in non-neutral and anisotropic conditions

PhD Thesis Mads Baungaard



#### PAPER • OPEN ACCESS

#### RANS simulation of a wind turbine wake in the neutral atmospheric pressure-driven boundary layer

M Baungaard<sup>1</sup>, M P van der Laan<sup>1</sup>, S Wallin<sup>2</sup> and M Abkar<sup>3</sup>

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Citation M Baungaard et al 2023 J. Phys.: Conf. Ser. 2505 012028

DOI 10.1088/1742-6596/2505/1/012028

# Outline



## **Computational fluid dynamics**



One method is not "better". They are just different tools.

# Computational cost?

Example: simulation of conventionally neutral boundary layer (CNBL)



#### **RANS** equations

Turbulence modelling: "How do we get the Reynolds stress tensor?"

$$\frac{\partial U_i}{\partial t} + \frac{\partial \left(U_j U_i\right)}{\partial x_j} = -\frac{1}{\rho} \frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} \left(\nu \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i}\right) - \overline{u'_i u'_j}\right) + F_i$$

Very important for getting good results!

#### Example 1: a wind turbine wake

Standard k-eps is over-diffusive.



#### Example 2: a row of turbines



#### Message

Before moving to expensive LES and DNS:

"What is the best we can do with RANS?"





# Linear eddy-viscosity models (EVMs)

#### How to model uiuj?



## How to model uiuj?



"Realism"

#### Linear eddy-viscosity models (EVMs)

$$\frac{\partial U_i}{\partial t} + \frac{\partial \left(U_j U_i\right)}{\partial x_j} = -\frac{1}{\rho} \frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} \left(\nu \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i}\right) - \overline{u'_i u'_j}\right) + F_i$$

Idea: Maybe the red term is similar to the blue term.

$$\Rightarrow \overline{u'_{i}u'_{j}} = -\nu_{t} \left( \frac{\partial U_{i}}{\partial x_{j}} + \frac{\partial U_{j}}{\partial x_{i}} \right) \qquad \qquad \begin{array}{l} \text{Symmetric} \\ \overline{u'_{i}u'_{i}} = 0 \end{array} \times$$

Idea 2: The trace should be equal to 2 times TKE

$$\boxed{\overline{u_i'u_j'} = -\nu_t \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i}\right) + \frac{2}{3}k\delta_{ij}} \qquad \begin{array}{l} \text{Symmetric} \\ \overline{u_i'u_i'} = 2k \\ \end{array}$$

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## The Boussinesq hypothesis

All linear EVMs use the Boussinesq hypothesis as their *constitutive relation*.

$$\overline{u_i'u_j'} = -\nu_t \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i}\right) + \frac{2}{3}k\delta_{ij}$$

They only differ in how they obtain the eddy viscosity.

$$\nu_{t} \equiv C_{\mu}^{(\text{eff})} k \tau \qquad \begin{array}{c} \text{Model} & C_{\mu}^{(\text{eff})} & \tau \\ \hline k \cdot \varepsilon & C_{\mu} & \frac{k}{\varepsilon} \\ k \cdot \varepsilon - f_{P} & C_{\mu} f_{P} & \frac{k}{\varepsilon} \\ k \cdot \omega & \beta^{*} & \frac{1}{\beta^{*} \omega} \\ \hline k \cdot \omega & \text{SST} & \min\left(\beta^{*}, \frac{a_{1}}{\sqrt{-2H_{\Omega}}F_{2}}\right) & \frac{1}{\beta^{*} \omega} \end{array}$$

#### What is meant by *linear* EVM?

$$\overline{u'_{i}u'_{j}} = -\nu_{t} \left( \frac{\partial U_{i}}{\partial x_{j}} + \frac{\partial U_{j}}{\partial x_{i}} \right) + \frac{2}{3}k\delta_{ij}$$

$$a_{ij} \equiv \frac{\overline{u'_{i}u'_{j}}}{k} - \frac{2}{3}\delta_{ij}, \qquad S_{ij} \equiv \frac{1}{2}\tau \left( \frac{\partial U_{i}}{\partial x_{j}} + \frac{\partial U_{j}}{\partial x_{i}} \right)$$

$$\nu_{t} \equiv C_{\mu}^{(\text{eff})}k\tau$$

$$\boxed{a_{ij} = -2C_{\mu}^{(\text{eff})}S_{ij}}$$

Anisotropy tensor is linear in the strain rate tensor.

# Deficiency of linear EVMs

$$a_{ij} = -2C_{\mu}^{(\text{eff})}S_{ij}$$

- "Blind" to the antisymmetric part of the velocity gradient tensor  $\tau \frac{\partial U_i}{\partial x_j} = S_{ij} + \Omega_{ij}$ which is important for flows with curvature, swirl or rotation.
- For horizontally homogeneous flows, linear EVMs give  $a_{11} = a_{22} = a_{33} = 0$
- Directions of  $a_{ii}$  and  $S_{ii}$  are always aligned
- Realisibility not ensured

Lots of ad-hoc modifications have been suggested to fix these problems!

...



# Theory of EARS models

Warning: lots of equations!

# How to model u<sub>i</sub>u<sub>j</sub>?



# The starting point for EARS models

• An exact equation



Terms in red need to be modelled

# A more compact form of the uiuj equation

• A typical decomposition

$$\underbrace{-\frac{1}{\rho}\overline{u_j'\frac{\partial p'}{\partial x_i}} - \frac{1}{\rho}\overline{u_i'\frac{\partial p'}{\partial x_j}}}_{vel \ pgrad \ corr} = \underbrace{\frac{1}{\rho}\overline{p'\left(\frac{\partial u_i'}{\partial x_j} + \frac{\partial u_j'}{\partial x_i}\right)}_{\Pi_{ij}} \underbrace{-\frac{1}{\rho}\left(\frac{\partial \overline{p'u_j'}}{\partial x_i} + \frac{\partial \overline{p'u_i'}}{\partial x_j}\right)}_{\mathcal{D}_{ij}^p}$$

and a typical model assumption

$$\varepsilon_{ij} = \frac{2}{3}\varepsilon\delta_{ij},$$

allow the uiuj equation to be written as:

$$\frac{D\overline{u_i'u_j'}}{Dt} = \mathcal{P}_{ij} + \mathcal{D}_{ij} + \Pi_{ij} - \frac{2}{3}\varepsilon\delta_{ij} \,.$$

# Differential Reynolds stress models (DRSMs)

$$\frac{D\overline{u_i'u_j'}}{Dt} = \mathcal{P}_{ij} + \mathcal{D}_{ij} + \Pi_{ij} - \frac{2}{3}\varepsilon\delta_{ij}$$

- 7 equations (6 for  $u_i u_j + 1$  for  $\varepsilon$ )
- Need models for

 $\Pi_{ij} \quad \mathcal{D}_{ij}^p \quad \mathcal{D}_{ij}^t$ 

 Complicated, but possible. Two models (LRR and SSG) are available in OpenFOAM.



# Example: rotating channel flow

• Velocity profile is asymmetric



• Turbulence activity also asymmetric



DNS from Grundenstam et al. (2008)

Launder et al. (1987)



# From differential RSM to algebraic RSM (ARSM)

Advantages of DRSMs:

More physics

Disadvantages of DRSMs:
X Expensive (7 eqs)
X Difficult to implement
X Numerical robustness not good

Idea by Rodi (1972,1976): *"Transform the DRSM equations into a set of algebraic equations."* 



# $\mathbf{a} = a_{ij} \equiv \frac{\overline{u'_i u'_j}}{k} - \frac{2}{3} \delta_{ij},$

# Derivation of the ARSM equations 1/3

• Rewrite DRSM equations to

$$\frac{Da_{ij}}{Dt} = \frac{1}{k} \left( \mathcal{P}_{ij} + \Pi_{ij} + \mathcal{D}_{ij} - \frac{2}{3} \varepsilon \delta_{ij} - \left( \frac{\overline{u'_i u'_j}}{k} \right) \left( \mathcal{P} + \mathcal{D}^{(k)} - \varepsilon \right) \right)$$

#### and rearrange

$$\Rightarrow \frac{Da_{ij}}{Dt} - \underbrace{\begin{pmatrix} \mathcal{D}_{j} & -\frac{u_{i}'u_{j}'}{k^{2}}\mathcal{D}^{(k)} \\ k & -\frac{u_{i}'u_{j}'}{k^{2}}\mathcal{D}^{(k)} \end{pmatrix}}_{\mathcal{D}_{ij}^{(a)}} = \frac{1}{k} \left( \mathcal{P}_{ij} + \Pi_{ij} - \frac{2}{3}\varepsilon\delta_{ij} - \left(\frac{\overline{u_{i}'u_{j}'}}{k}\right)(\mathcal{P} - \varepsilon) \right)$$

#### Derivation of the ARSM equations 2/3

• The "weak-equilibrium assumption" gives an *algebraic* set of equations

$$0 = \mathcal{P}_{ij} + \Pi_{ij} - \frac{2}{3}\varepsilon\delta_{ij} - \left(\frac{\overline{u'_i u'_j}}{k}\right)\left(\mathcal{P} - \varepsilon\right)$$

and rewrite to

$$a_{ij}\left(\frac{\mathcal{P}}{\varepsilon}-1\right) = \frac{\mathcal{P}_{ij}}{\varepsilon} + \frac{\Pi_{ij}}{\varepsilon} - \frac{2}{3}\frac{\mathcal{P}}{\varepsilon}\delta_{ij}$$

#### Derivation of the ARSM equations 3/3

$$a_{ij}\left(\frac{\mathcal{P}}{\varepsilon}-1\right) = \frac{\mathcal{P}_{ij}}{\varepsilon} + \frac{\Pi_{ij}}{\varepsilon} - \frac{2}{3}\frac{\mathcal{P}}{\varepsilon}\delta_{ij}$$

• Shear production definitions (exact)

$$\frac{\mathcal{P}_{ij}}{\varepsilon} = -a_{kj} \left( S_{ik} + \Omega_{ik} \right) - a_{ik} \left( S_{kj} - \Omega_{kj} \right) - \frac{4}{3} S_{ij} \qquad \qquad \frac{\mathcal{P}}{\varepsilon} = -a_{ik} S_{ki}$$

• Launder-Reece-Rodi (LRR) pressure-strain model

$$\frac{\Pi_{ij}}{\varepsilon} = -c_1 a_{ij} + \frac{4}{5} S_{ij} + \frac{9c_2 + 6}{11} \left( a_{ik} S_{kj} + S_{ik} a_{kj} - \frac{2}{3} a_{km} S_{mk} \delta_{ij} \right) + \frac{7c_2 - 10}{11} \left( a_{ik} \Omega_{kj} - \Omega_{ik} a_{kj} \right)$$

EARSM theory

#### **LRR-ARSM**

$$\left(\frac{\mathcal{P}}{\varepsilon} - 1 + c_1\right)\mathbf{a} = -\frac{8}{15}\mathbf{S} + \frac{9c_2 - 5}{11}\left(\mathbf{aS} + \mathbf{Sa} - \frac{2}{3}\mathrm{tr}\left\{\mathbf{aS}\right\}\mathbf{I}\right) + \frac{7c_2 + 1}{11}\left(\mathbf{a\Omega} - \mathbf{\Omega}\mathbf{a}\right)$$

Simplification used by Taulbee (1992).  $\longrightarrow$  If  $c_2=5/9$  DNS by Shih & Shabbir (1993) support it.

Simplified LRR-ARSM 
$$\underbrace{\frac{9}{4}\left(\frac{\mathcal{P}}{\varepsilon}-1+c_{1}\right)}_{N}\mathbf{a}=-\frac{6}{5}\mathbf{S}+(\mathbf{a}\boldsymbol{\Omega}-\boldsymbol{\Omega}\mathbf{a})$$

EARSM theory

# Simplified LRR-ARSM

$$N\mathbf{a} = -\frac{6}{5}\mathbf{S} + (\mathbf{a}\mathbf{\Omega} - \mathbf{\Omega}\mathbf{a})$$

Only assumptions:

- Isotropic dissipation
- Weak-equilibrium assumption
- The LRR pressure-strain model with c<sub>2</sub>=5/9

#### Problems:

- 1. Implicit equation for **a**
- 2. Non-linear in a



# From algebraic RSM to explicit algebraic RSM

J. Fluid Mech. (1975), vol. 72, part 2, pp. 331–340 Printed in Great Britain

A more general effective-viscosity hypothesis

By S. B. POPE

Idea of Pope (1975):

- 1. Expand anisotropy tensor
- 2. Insert expansion in ARSM and simplify
- 3. Solve for expansion coefficients

In the mid-90s: (Optional 4.) Ensure self-consistency



# Step 1: anisotropy expansion

$$a_{ij} = \sum_{l=1}^{\infty} \beta_l T_{ij}^{(l)}$$

#### The anisotropy tensor must be:

- Dimensionless
- Galilean invariant
- Symmetric
- Traceless
- ..and so must the basis tensors too.

A complete tensor basis for statistically two-dimensional flows:

$$\mathbf{a} = \beta_1 \underbrace{\mathbf{S}}_{\mathbf{T}^{(1)}} + \beta_2 \underbrace{\left(\mathbf{S}^2 - \frac{1}{3} \mathrm{tr}\{\mathbf{S}^2\}\right)}_{\mathbf{T}^{(2)}} + \beta_4 \underbrace{\left(\mathbf{S}\Omega - \Omega\mathbf{S}\right)}_{\mathbf{T}^{(4)}}$$

Basis candidates	Sym.	Tr. less
$\mathbf{S}$	<ul> <li>✓</li> </ul>	<ul> <li>Image: A set of the set of the</li></ul>
$\Omega$	×	<ul> <li>✓</li> </ul>
$\mathbf{S}^2$	<ul> <li>✓</li> </ul>	×
$\mathbf{S}^2 - rac{1}{3} \mathrm{tr} \{ \mathbf{S}^2 \} \mathbf{I}$	~	<ul> <li>✓</li> </ul>
$\mathbf{\tilde{S}}\mathbf{\Omega}$	×	<ul> <li>✓</li> </ul>
${f S}\Omega-\Omega{f S}$	<ul> <li>✓</li> </ul>	<b>v</b>

#### Step 2: insert and simplify

$$N\mathbf{a} = -\frac{6}{5}\mathbf{S} + (\mathbf{a}\mathbf{\Omega} - \mathbf{\Omega}\mathbf{a})$$

Insert expansion of **a** (trivial)

$$N\left(\beta_{1}\mathbf{T}^{(1)} + \beta_{2}\mathbf{T}^{(2)} + \beta_{4}\mathbf{T}^{(4)}\right) = -\frac{6}{5}\mathbf{S} + \left(\beta_{1}\mathbf{T}^{(1)} + \beta_{2}\mathbf{T}^{(2)} + \beta_{4}\mathbf{T}^{(4)}\right)\mathbf{\Omega} - \mathbf{\Omega}\left(\beta_{1}\mathbf{T}^{(1)} + \beta_{2}\mathbf{T}^{(2)} + \beta_{4}\mathbf{T}^{(4)}\right)$$
  

$$\int \mathbf{Simplify RHS (non-trivial)}$$
  

$$N\left(\beta_{1}\mathbf{T}^{(1)} + \beta_{2}\mathbf{T}^{(2)} + \beta_{4}\mathbf{T}^{(4)}\right) = \left(-\frac{6}{5} + 2\beta_{4}II_{\Omega}\right)\mathbf{T}^{(1)} + \beta_{1}\mathbf{T}^{(4)}$$

Step 3: finding the coefficients  

$$N\left(\beta_{1}\mathbf{T}^{(1)} + \beta_{2}\mathbf{T}^{(2)} + \beta_{4}\mathbf{T}^{(4)}\right) = \left(-\frac{6}{5} + 2\beta_{4}II_{\Omega}\right)\mathbf{T}^{(1)} + \beta_{1}\mathbf{T}^{(4)}$$
Equate coefficients  

$$N\beta_{1} = \left(-\frac{6}{5} + 2\beta_{4}II_{\Omega}\right), \qquad \text{Assume } N \text{ is known} \qquad \beta_{1} = -\frac{6}{5}\frac{N}{N^{2} - 2II_{\Omega}},$$

$$N\beta_{2} = 0, \qquad \beta_{4} = -\frac{6}{5}\frac{1}{N^{2} - 2II_{\Omega}}.$$

$$\beta_{4} = -\frac{6}{5}\frac{1}{N^{2} - 2II_{\Omega}}.$$

$$\mathbf{a} = -\frac{6}{5}\frac{N}{N^{2} - 2II_{\Omega}}\mathbf{T}^{(1)} - \frac{6}{5}\frac{1}{N^{2} - 2II_{\Omega}}\mathbf{T}^{(4)}}{\beta_{4}}$$

#### Step 4: self-consistency (1/2)

$$\mathbf{a} = \underbrace{-\frac{6}{5} \frac{N}{N^2 - 2II_{\Omega}}}_{\beta_1} \mathbf{T}^{(1)} \underbrace{-\frac{6}{5} \frac{1}{N^2 - 2II_{\Omega}}}_{\beta_4} \mathbf{T}^{(4)}$$

- "Self-consistency" = No approximations from ARSM to EARSM.
- $\rightarrow$  We need a N, such that the above equation satisfies the simplified ARSM.

$$N \equiv \frac{9}{4} \left( \frac{\mathcal{P}}{\varepsilon} - 1 + c_1 \right)$$
$$= \frac{9}{4} \left( -\text{tr}\{\mathbf{aS}\} - 1 + c_1 \right)$$
$$= \frac{9}{4} (c_1 - 1) + \frac{27}{10} \frac{NII_S}{N^2 - 2II_\Omega}$$

 $\rightarrow$  A cubic polynomial for N

#### Step 4: self-consistency (2/2)

Johansson & Wallin (1996): the real and positive root is

$$N = \begin{cases} \frac{c_1'}{3} + \left(P_1 + \sqrt{P_2}\right)^{1/3} + \operatorname{sign}\left(P_1 - \sqrt{P_2}\right)|P_1 - \sqrt{P_2}|^{1/3}, & P_2 \ge 0\\ \frac{c_1'}{3} + 2\left(P_1^2 - P_2\right)^{1/6} + \cos\left(\frac{1}{3}\left(\frac{P_1}{\sqrt{P_1^2 - P_2}}\right)\right), & P_2 < 0 \end{cases}$$

$$P_{1} = \left(\frac{1}{27}c_{1}^{\prime 2} + \frac{9}{20}II_{S} - \frac{2}{3}II_{\Omega}\right)c_{1}^{\prime},$$
$$P_{2} = P_{1}^{2} - \left(\frac{1}{9}c_{1}^{\prime 2} + \frac{9}{10}II_{S} + \frac{2}{3}II_{\Omega}\right)^{3}.$$

# Summary of the model

• The 2D EARS model of Wallin & Johansson (2000) with *k*-ε platform.

$$P_{1} = \left(\frac{1}{27}c_{1}^{\prime 2} + \frac{9}{20}II_{S} - \frac{2}{3}II_{\Omega}\right)c_{1}^{\prime},$$

$$P_{2} = P_{1}^{2} - \left(\frac{1}{9}c_{1}^{\prime 2} + \frac{9}{10}II_{S} + \frac{2}{3}II_{\Omega}\right)^{3}.$$

$$N = \begin{cases} \frac{c_{1}^{\prime}}{3} + (P_{1} + \sqrt{P_{2}})^{1/3} + \operatorname{sign}\left(P_{1} - \sqrt{P_{2}}\right)|P_{1} - \sqrt{P_{2}}|^{1/3}, \quad P_{2} \ge 0\\ \frac{c_{1}^{\prime}}{3} + 2\left(P_{1}^{2} - P_{2}\right)^{1/6} + \cos\left(\frac{1}{3}\left(\frac{P_{1}}{\sqrt{P_{1}^{2} - P_{2}}}\right)\right), \qquad P_{2} < 0 \end{cases}$$

$$\boxed{5} \quad \beta_{1} = -\frac{6}{5}\frac{N}{N^{2} - 2II_{\Omega}}, \qquad \beta_{4} = -\frac{6}{5}\frac{1}{N^{2} - 2II_{\Omega}}.$$

$$\boxed{6} \quad a_{ij} = \beta_{1}T_{ij}^{(1)} + \beta_{4}T_{ij}^{(4)}$$

$$\boxed{7} \quad \overline{u_{i}^{\prime}u_{j}^{\prime}} = ka_{ij} + \frac{2}{3}k\delta_{ij}$$

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## Implementation

- Easy to implement in a code, which already has a two-equation model implemented.
- Trick: split the anisotropy tensor



• Remember to change shear production in transport equations

$$\mathcal{P} = \nu_t \left( \frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i} \right) \frac{\partial U_i}{\partial x_j}$$

$$\mathcal{P} = -\varepsilon a_{ik} S_{ki}.$$



# Application of EARS models

# Codes with the WJ-EARS model

- <u>Maple/FORTRAN 1D code</u> (KTH)
- Python 1D code (KTH)
- <u>Edge</u> (FOI/KTH)
- <u>FINFLO</u> (Helsinki University)
- <u>TAU</u> (DLR)
- <u>EllipSys1D and EllipSys3D</u> (DTU)
- Ansys CFX (<u>manual</u>)
- OpenFOAM user model (<u>implemented for OF1.7.x</u>)

•

...

# Square duct flow (1/3)

• Secondary flow in corners



# Square duct flow (2/3)

• Simulation of lower left quadrant in EllipSys3D with WJ-EARS model



# Square duct flow (3/3)



#### A neutral ABL (1/2)

$$k \equiv \frac{1}{2}(\overline{u'u'} + \overline{v'v'} + \overline{w'w'})$$

What is the distribution of TKE among its components in the neutral atmospheric boundary layer (ABL)?



#### A neutral ABL (2/2)



Impossible to capture with linear EVMs!

#### A neutral ABL + a wind turbine (1/4)



- EllipSys3D code
- V80 turbine modelled as AD
- Grid stretching to avoid excessive amount of cells

# A neutral ABL + a wind turbine (2/4)

• Comparison of disk-averaged quantities with LES data



# A neutral ABL + a wind turbine (3/4)

Why does k-ε overpredict wake recovery?
 Because shear stress is overpredicted.



# A neutral ABL + a wind turbine (4/4)

• Linear EVM (k- $\varepsilon$ - $f_p$ ) has too isotropic wake turbulence





# Takeaways

#### What is an EARS model?

- A turbulence model derived directly from the Reynolds stress equations.
- Does not rely on the Boussinesq hypothesis!

#### Why use an EARS model?

- Relatively easy to implement.
- ✓ Only slightly more computationally expensive than linear EVMs.
- Automatically includes more physics than linear EVMs.
- ✓ More numerically robust than DRSMs.

# The end

#### A good introduction (4 pages):

A new explicit algebraic Reynolds stress model

Authors Arne V Johansson, Stefan Wallin

Publication date 1996/7/2

Book Advances in Turbulence VI: Proceedings of the Sixth European Turbulence Conference, held in Lausanne, Switzerland, 2–5 July 1996 https://link.springer.com/chapte r/10.1007/978-94-009-0297-8\_8

Some things are easier to discuss at a blackboard! We can talk more about EARS models:

> Wednesday 7 Aug @ 15.00 @ Lecture room 7

https://mchba.github.io/240731 intro to earsm.pdf



Slides are here!

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# Extra slides

# Realisability: limits of the anisotropy tensor



Greek indices imply that there is <u>no</u> summation over repeated indices!

- Rule 1: Each normal stress must be positive.
- Rule 2: From the definition of TKE.
- Rule 3: Each shear stress must satisfy the Cauchy-Schwarz inequality.
- Rule 4: Comes from rule 2.

## k-eps can give unrealizable turbulence



# Example 2: swirling flow in an expanding pipe



Hanjalic & Launder (2011, p.81)

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# Rotating homogeneous shear flow

• A simple first testcase



### A convective ABL

Zeli et al. (2021)



**Applications**